

# Integrated Biomass and Solar Thermal Power Plant - A Case Study

Gade Mahesh Babu, T.Srinivas, K.V.R.Naidu

**Abstract**— Renewable energy technologies are clean and meet the energy demand which is not met by conventional sources of energy technologies, The cost of Linear Fresnel Reflector (LFR) technology is low compared to other solar concentrating collectors and suitable for medium temperature heat recovery. The work focuses on experimental development and performance evaluation of LFR with a single axis tracking mode and integration of it with biomass power plant. In LFR collector, the sunlight is concentrated on a fixed absorber tube to heat the working fluid which passes through the tube and the heated working fluid is used for thermal applications. Six number of thin flat plane reflective mirrors with high reflectivity have been used to reflect the solar radiation on a focal line. There are three round copper tubes used as fixed receiver tubes placed at the focal line to absorb the concentrated heat. Although the current experimental model is small, it will bring a breakthrough of commercial solar energy concentration application technology in the near future, because of its low cost compared to parabolic trough collector. The thermal performance of the LFR solar device is studied experimentally during peak sunny hours. By integrating this technology with biomass power plant is attractive for their potential to increase power output, decrease dependence on Biomass fuels and to reduce greenhouse gases. The main objective of this work is to find the efficiency of LFR at VIT University, Vellore conditions. This work also extended to develop a biomass power plant with the integration of solar reflectors.

**Index Terms**— Biomass combustion, Biomass power plant, Concentrator devices, Integrated energy systems, linear fresnel reflectors, Renewable energy, solar thermal energy.

---

## 1 INTRODUCTION

### 1.1 Linear Fresnel Reflectors

Linear fresnel reflectors consists of long flat mirrors fixed horizontally on base [1]. Each mirror is tilted at an angle so that all incident sun rays falling on mirrors are reflected to a same focus. The absorber is placed on the focus so that the concentrated radiation is absorbed by the absorber. The absorber is nothing but a series of tubes or tubes which contains a heat transfer fluid. Absorber plays a very important role in the solar concentrating device for the collection of solar energy. In the solar collectors, Heat loss from the absorber occurs through convection and conduction and radiation modes [2], [3], [4], [5].

The solar collectors get heated up due to the incident radiation and then it emits radiation. Due to this radiation there will be heat loss from the absorber tube by which there will be decrease in thermal efficiency of the collector. Heat losses from the absorber should be less, to get maximum efficiency. The absorber should be a high thermal conductive material so that the radiation loss will be decreased. The absorber covered with glass cover to reduce the thermal losses. The non-evacuated solar absorber painted with ordinary black paint and covered with plane glass, yielded poor performance [6],[7]. Sharma et al found maximum convective heat transfer in the receiver tube when fluid flow was turbulent. Thermal

performance of the linear fresnel concentrator was studied with a fixed concentration ratio. Linear fresnel reflector devices [8] have many advantages when compared with parabolic trough collectors, (i) it is useful for medium temperature range (100–250°C) applications [9] (ii) it is designed with linear flat mirrors and constituent materials for its fabrication are available in the market; (iii) the linear shape and the distances between the adjacent flat mirrors result in very less wind loading on the reflectors. So, it can be mounted on simple cost-effective supporting structure.

### 1.2 Biomass

The potential for biomass boilers in India is vast with over 370 million tons of biomass being produced every year. Biomass is available from agricultural wastes, direct harvesting and as a by-product from industries such as rice mills, sugar mills and saw mills. However, due to problems with infrastructure and the seasonal variability of biomass in India, consumers are struggling to obtain a consistent fuel supply. Furthermore, while biomass is still competitive, prices have increased considerably in recent years [10].

Hybridization of solar thermal with biomass combines two energy sources that complement each other, both seasonally and diurnally, to overcome their individual drawbacks [11]. During the day the sun's rays can be harnessed by solar collectors and biomass feedstock can be burnt as a supplementary fuel to achieve constant base load operation. CSP plants benefit from hybridization or effective energy storage due to the variable nature of solar energy, particularly in India's monsoon season. Constant

---

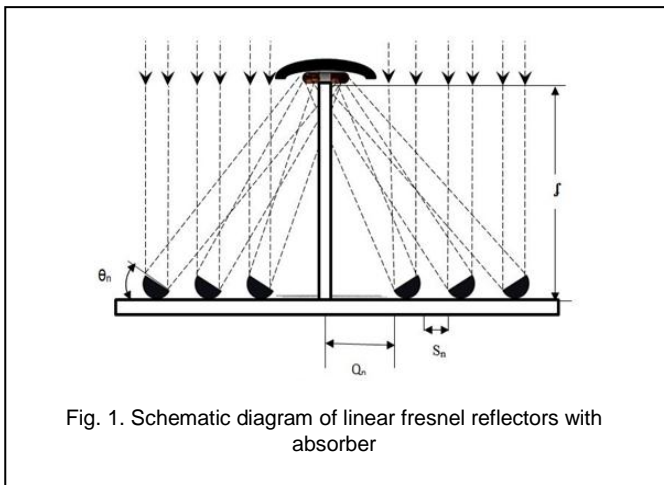
• Gade Mahesh babu is currently pursuing masters degree program in Energy & Environmental Engineering in VIT University, Vellore, India, PH-09652307060. E-mail: [mahesh.babu397@mail.com](mailto:mahesh.babu397@mail.com)

• T.Srinivas is Professor in CO<sub>2</sub> Research and Green Technologies Centre in VIT University, Vellore, India, PH- 8056300923. E-mail: [srinivaspalli@vit.ac.in](mailto:srinivaspalli@vit.ac.in)

base load or full load plants are typically implemented as plant efficiency is maximized and unit cost of energy is minimized. However, solar energy could be used to increase plant output during the day [12]. In comparison to a biomass-only system, solar hybridization reduces biomass demand, thus improving energy security and decreasing land required for farming and storage [13].

## 2. DESIGN CONSIDERATION FOR LINEAR FRESNEL REFLECTING CONCENTRATOR

The linear fresnel reflectors have mainly two elements: (i) Reflectors, (ii) Absorber. Fig. 1. Shows the schematic of linear fresnel reflecting concentrator with absorber. Some assumptions are made to design the linear fresnel reflecting solar concentrator were: (i) the reflectors are perfectly tracked according movement of the sun from East to West, (ii) the mirror elements are capable of reflecting light, every mirror element were considered with equal width ( $W$ ). The tilt of each mirror element was adjusted so that the incident light rays to the aperture area [16], reached the focus point 'F' after one reflection. A distance (called shift) is maintained between adjacent mirrors so that a mirror shadow will not fall on the other [17].

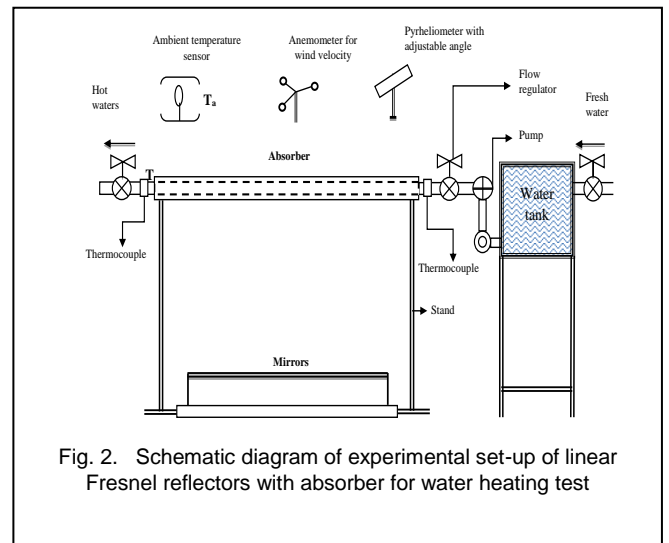


Every mirror is distinguished by three parameters: tilt ( $\theta_n$ ), shift ( $S_n$ ) and location ( $Q_n$ ), as shown in Fig.1.

## 3. EXPERIMENTAL SET-UP

The schematic diagram of experimental set up is shown in Fig. 2. It consisted of one tank, absorber set up, reflector set up, flow regulator, pump, thermocouples, etc. The tank is filled with water. The absorber set up consists of three copper pipes which are connected in series with L-bends. The reflectors set up consist of 6 linear reflectors which are placed in parallel. The water tank was kept at a position and connected to absorber set up. The water in the tank was passed through the absorber at a desired flow rate with help of pump. The absorber pipe was connected with tank by a tube. A flow regulator valve was fitted to control the

water flow through the absorber pipe. A small pump (50 W capacity) was used to pump from water tank to outlet valve. Thermal performance of three set of similar absorbers with round pipe for linear fresnel solar concentrating collector were studied in the laboratory conditions. The details of the trapezoidal cavity absorbers are given below. Schematic diagram of experimental set up of linear fresnel reflectors is shown in Fig. 2. Absorber pipe was made of a set of three copper round tubes (outer diameter: 12.7 mm, thickness: 1.5 mm and length: 1000 mm).



Overall heat loss coefficient of the absorber ( $UL$ ) was studied in the experiment. Observations were taken at different fluid temperatures on steady state condition. The average of the inlet and outlet fluid temperatures is estimated as absorber temperatures. The average air flow rate at the bottom of the absorber was measured by anemometer it was on an average of 2 m/s during the study.

## 4. TEST PROCEDURE

### 4.1. Thermal Efficiency of Reflector - Absorber System

The linear reflecting solar device with absorber was tested outdoors at VIT University, Vellore (12.980 N and 79.1330 E). The reflectors were arranged in North - South directions horizontal and East - West sun tracking Fig 3 shows the fabricated linear fresnel reflector device and absorber. The thermal efficiency test for reflector absorber system was conducted by maintaining the same mass flow rate of the heat transfer fluid (water) flowing through the absorber pipe with the help of flow regulator. A record of inlet ( $T_i$ ), outlet ( $T_o$ ) fluid temperatures, ambient temperature ( $T_a$ ), solar beam intensity ( $I_b$ ) at several intervals of time on the absorber and the mass flow rate of the fluid is made.

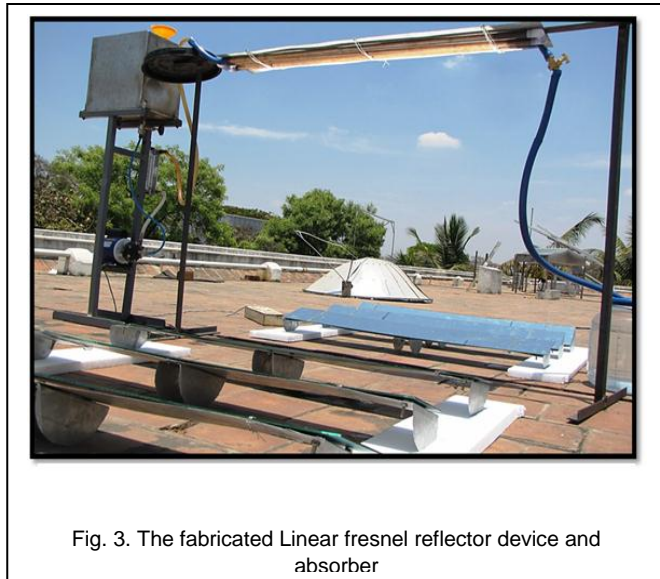


Fig. 3. The fabricated Linear fresnel reflector device and absorber

Thermal efficiency ( $\eta_{th}$ ) is calculated from following formula:-

$$\eta_{th} = \frac{mC_p(T_o - T_i)}{I_b \times A_a} \times 100 \quad (1)$$

Where m and  $C_p$  are the mass flow rate and the specific area of the aperture of the reflector, and  $T_i$  and  $T_o$  are the inlet and outlet water temperatures respectively. Calibrated thermocouples insulated which were used for measuring the temperature of water at the inlet and outlet of the absorber. These thermocouples were connected to a digital thermometer with display to record their outputs. Pyrheliometer is connected to a multimeter for recording beam radiation [18]. Pyrheliometer is calibrated according to sun position before taking the readings. Table 1. Shows the tabulated values.

## 5. WORKING OF INTEGRATED BIOMASS AND SOLAR POWER PLANT

The agricultural waste such as Rice husk is transported to the power plant for storage .it is then supplied to fuel handing system where it is crushed into small pieces by shredders. The crushed fuel is then fed into the boiler through belt and screw conveyers [19] .The paddy husk is temporarily stored in the bunker though belt conveyers

### 5.1 Biomass power plant

In the boiler the fuel is burned and the water from steam drum is converted in to high pressure steam which is

further superheated in primary and secondary super heater [20] .The super-heated steam is passed through the steam turbine at 485°C and 61 Kg/cm<sup>2</sup> thus the turbine converts heat energy in to mechanical energy and then by generator the electricity is generated at 4Mwh. Fig. 4. Schematic diagram of 4.5 MW biomass power plant.

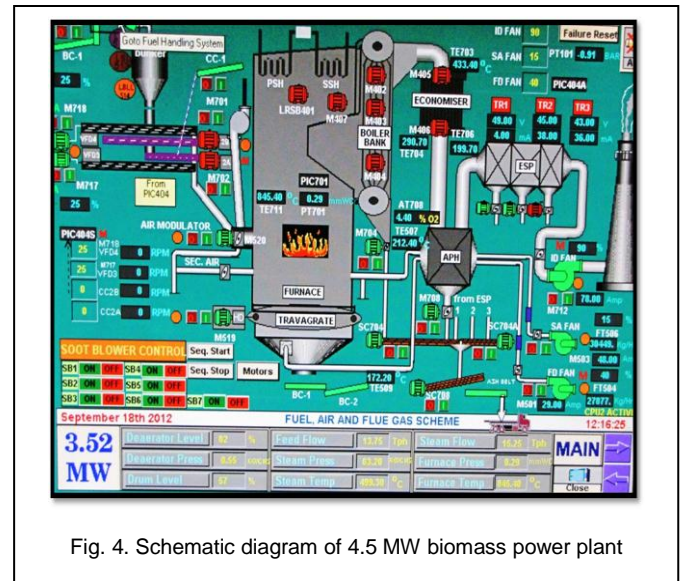
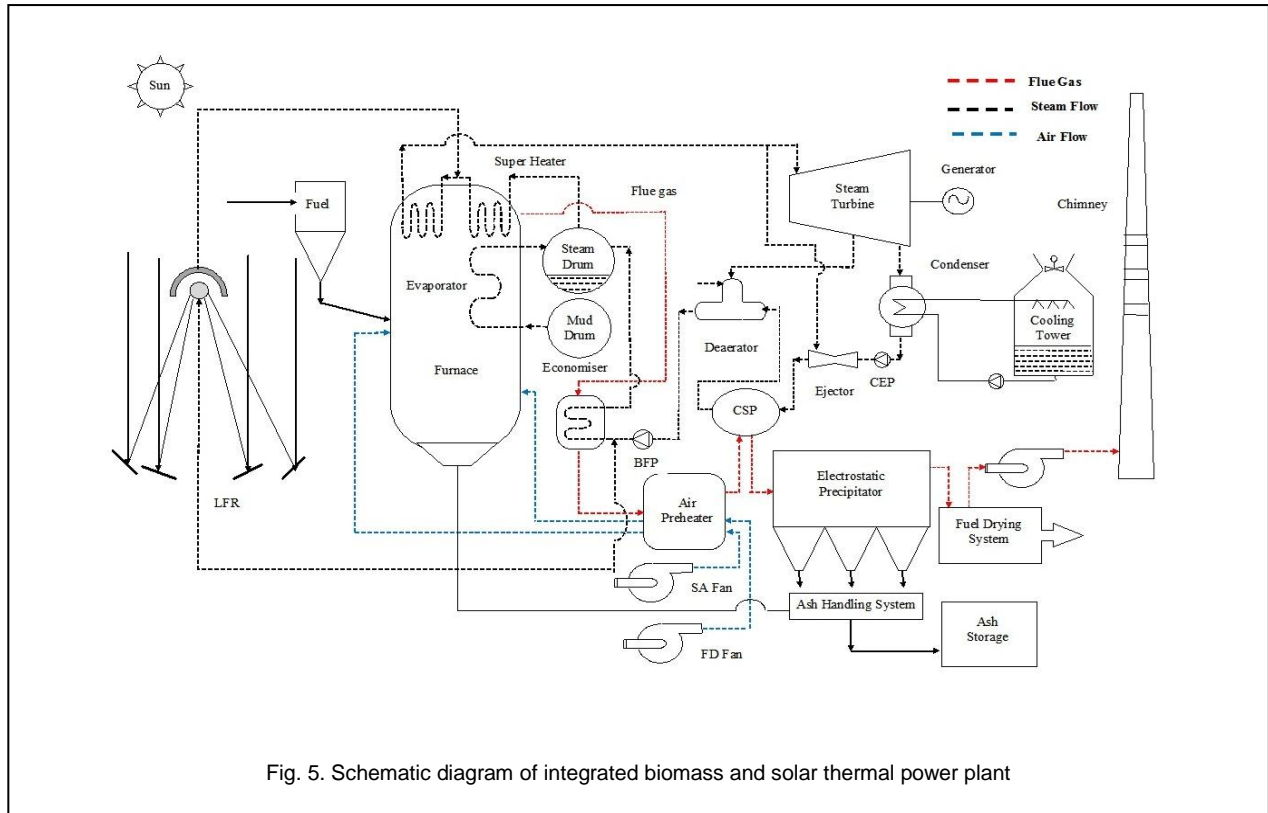


Fig. 4. Schematic diagram of 4.5 MW biomass power plant

The pressure of steam decreases and its volume increases it is then passes out from steam turbine in to the condenser where the condensate is produced the condensate is extracted by Condensate Extraction Pump (CEP) and passed through ejector. The water required for steam generation is supplied to deaerator from ejector. The excess water required for steam generation is supplied from water treatment plant to deaerator to maintain the water level and it is maintained at above 100°C to remove the air from the water, then it is passes through economiser by Boiler Feed pump (BFP) to increases the temperature of steam .The steam from economiser is passed through evaporator by steam drum and mud drum. The steam in the condenser is cooled by supplied cooled water from the cooling tower. In condenser the cooled water is further supplied cooled water circulate with the help of pump which condenses the large pressure wet steam.

### 5.2 Integration of LFR in biomass power plant

As show in the Figure 5 the LFR is parallel integrated in the biomass power plant .The water from deaerator is splitted after BFP to LFR system where the water is heated by solar energy as shown in the LFR experimentation, after that the heated water is joined at super heater and the cycle continues as shown in the fig. 5.



## 6. RESULTS AND DISCUSSION

The results of the efficiency measurement are presented in Table.1.the average thermal efficiency of the concentrator is 68.61%. This is due to the good reflectivity of the six mirror elements. The readings are taken when the sun is at zenith. The use of three copper pipes as absorber is an advantage because of its high thermal conductivity and more absorber area. Fig.6 shows variation of efficiency with time based on aperture area, total used area of reflector system. But there are heat losses by conduction, convection, radiation due to no glass covers.

The efficiency of linear fresnel reflector system is more than parabolic trough collector and cost of this device is around Rs 3000 which very much less than parabolic trough collector. The tracking of adjusted manually according to the position of sun. Fig. 7, Fig. 8 shows the variation of efficiency and based on the total area of system and reflective area

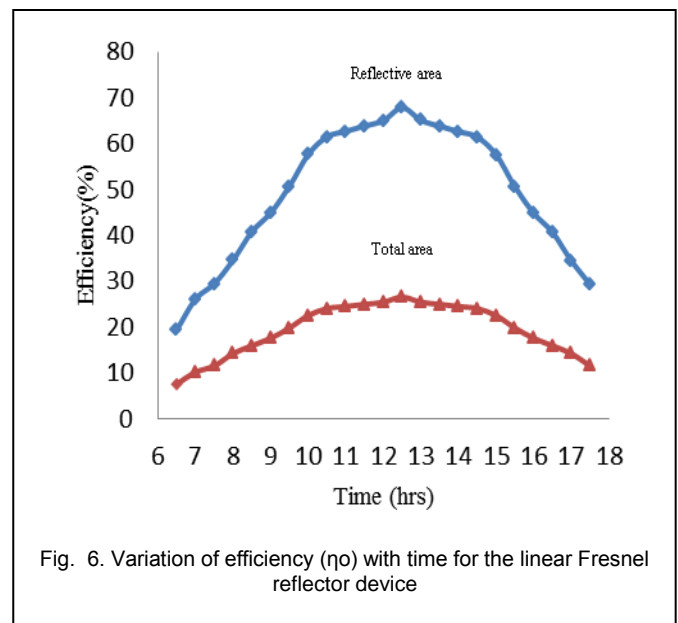


TABLE 1  
THERMAL EFFICIENCY ( $\eta_{th}$ ) OF THE LINEAR FRESNEL REFLECTOR

Time (hr)	Intensity of beam radiation $I_b$ (W/m <sup>2</sup> )	Ambient temperature ( $T_a$ )°C	Inlet temperature ( $T_i$ )°C	Outlet temperature ( $T_o$ )°C	Mass flow rate of the water $m$ (kg/sec)	Thermal efficiency ( $\eta_{th}$ )%	Average thermal efficiency %
6.5	239.909	25.9	26	27	0.01	19.377	
7	268.134	26.5	27	28.5	0.01	26.110	
7.5	268.134	27.1	27.5	29.2	0.01	29.470	
8	282.247	27.7	27.9	30	0.01	34.580	
8.5	296.359	27.9	28.1	30.7	0.01	40.780	
9	310.471	28.5	28.9	31.9	0.01	44.920	
9.5	311.882	28.8	29	32.4	0.01	50.680	
10	314.705	29	29.2	33.1	0.01	57.610	
10.5	317.529	29	29.3	33.5	0.01	61.490	
11	318.999	30	30.1	34.4	0.01	62.670	49.367
11.5	320.350	30	30.2	34.6	0.01	63.850	
12	321.761	30	30.3	34.8	0.01	65.010	
12.5	429.370	30.1	30.32	36.5	0.01	67.82	
13	321.761	30	30.3	34.8	0.01	65.010	
13.5	320.350	30	30.2	34.6	0.01	63.850	
14	318.999	30	30.1	34.4	0.01	62.630	
14.5	317.527	29	29.3	33.5	0.01	61.420	
15	314.705	29	29.2	33.1	0.01	57.59	
15.5	311.882	28.8	29	32.4	0.01	50.610	
16	310.471	28.5	28.9	31.9	0.01	44.850	
16.5	296.359	27.9	28.1	30.7	0.01	40.69	
17	282.247	27.7	27.9	30	0.01	34.480	
17.5	268.134	27.1	27	29.2	0.01	29.320	

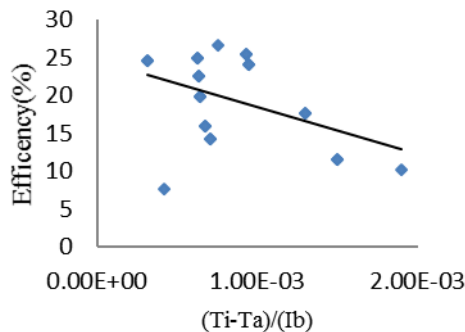


Fig. 7. Variation of efficiency ( $\eta_o$ ) with  $(T_i - T_a)/(I_b)$  based on Total area

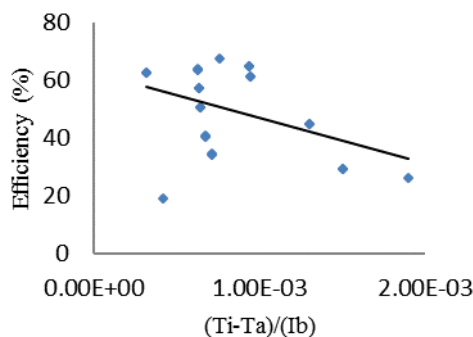


Fig. 8. Variation of efficiency ( $\eta_o$ ) with  $(T_i - T_a)/(I_b)$  based on Reflective area

### 7. Conclusion

These are several merits due to integration of LFR with Biomass power plant i.e. increase in power output, thermal efficiency, exergy efficiency. Improvement with integration of Linear Fresnel reflectors to rise water temperature. Improvement in performance with fuel drying Control in fuel feed rate with variable solar radiation, Optimized biomass usage is maintained.

TABLE 2  
Nomenclature

Symbol	Quantity
$A_s$	surface area of the absorber pipe (m <sup>2</sup> )
$A_c$	aperture area of the fresnel solar reflecting concentrator (m <sup>2</sup> )
$C_p$	specific heat of the fluid (kJ/kg°C)
CR	concentration ratio
$h$	height of the absorber above primary mirror reflector plane (m)
$I_b$	Solar beam intensity normal to the aperture plane (kW/m <sup>2</sup> )
$m$	fluid mass flow rate (kg/s)
$T_i$	inlet fluid temperature (°C)
$T_o$	outlet fluid temperature (°C)
$T_a$	ambient temperature (°C)
UL	overall heat loss coefficient (W/m <sup>2</sup> °C)
$Q_n$	distance between the n <sup>th</sup> mirror and center (m)
$S_n$	shift or space between n <sup>th</sup> and n <sup>th</sup> -1 mirror (m)
$\theta_n$	tilt of the nth mirror
$\eta_o$	instantaneous thermal efficiency of the fresnel reflecting solar concentrator-receiver

$m$  = meter,  $J$  = joule,  $T$  = temperature,  $kg$  = kilogram, .

## ACKNOWLEDGMENTS

Our sincere regards to K.Balasubramanyam, Chinna, Matrix Power Pvt Ltd for their data support. My sincere thanks to Mr.Ravibabu for his valuable guidance in the field of environmental. Dedication of this work to Gade Rajendra Prasad, Gade Malliswari, Gade Deepthi, Gade Divya and well-wishers.

## REFERENCES

- [1] Mathur SS, Kandpal TC, Negi BS, Study of concentration characteristics and performance evaluation of a linear fresnel reflector, National Solar Energy Conference, New Delhi: Tata McGraw Hill publishing Co.; 1988. p. 263–70.
- [2] Duffie JA, Beckman WA. Solar Energy of thermal processes, New York: John Wiley; 1991.
- [3] Dey CJ, Heat transfer aspect of an elevated linear absorber, Solar Energy 2004.p.243–9.
- [4] Reynolds DJ, Jance MJ, Behnai M, Morrison GL. An experimental and computational study of the heat loss characteristics of a trapezoidal cavity absorber. Sol Energy 2004; 76:229–34.
- [5] Doron P, Kribus A. The effect of irradiation directional distribution on absorption in volumetric solar receivers. J Sol Energy Eng 1997; 119:68–73.
- [6] Wackelgard E. Characterization of black nickel solar absorber coatings electroplated in a nickel chlorine aqueous solution. Sol Energy Mater Sol Cells 1998; 56:35–44.
- [7] John S, Nagarani N, Rajendran S. Black cobalt solar absorber coatings. Sol Energy Mater 1991; 22:293–302.
- [8] Hsieh JS. Solar energy engineering. Englewood Cliffs, New Jersey: Prentice-Hall Inc.; 1986.
- [9] Singh PL, Sarviya RM, Bhagoria JL. Improvement in thermal performance of the absorber for linear fresnel solar concentrating collector. In: Proc. of the 15th ISME Int. conf. on new horizons of mechanical eng. New Delhi: Elite Publishing House Pvt. Ltd.; 2008. p. 252–55.
- [10] Khan MdKA. Copper oxide coating for use in a linear solar fresnel reflecting concentrating collector. Renew Energy 1999; 7:603–8.
- [11] Singh PL, Ganesan S, Yadava GC. Performance of a linear fresnel concentrating solar device. Renew Energy 1999; 18:409–16.
- [12] E.D. Larson, A review of biomass integrated-gasifier/gas turbine combined cycle technology and its application in sugarcane industries, with an analysis for Cuba, Energy Sust Dev, 5 (2001), pp. 54–76
- [13] Sharma MS, Mathur SS, Singh RN. Performance analysis of a linear solar concentrator under different flow regimes. Appl Energy 1983; 13(1):77–81.
- [14] Anuradha Mishra, M N chakravarty and N d Kaushika . Thermal optimization of solar biomass hybrid cogeneration plants. Journal of scientific & Industrial Research. 2006, pp. 355-363
- [15] T. Srinivas, A. V. S. S. K. S. Gupta, B. V. Reddy. Thermodynamic Equilibrium Model and Exergy Analysis of a Biomass Gasifier. Journal of Energy Resources Technology. September 2009. Vol. 131
- [16] Negi BS, Mathur SS, Kandpal TC. Thermal and thermal performance evaluation of a linear fresnel reflector solar concentrator. Sol Wind Technol 1989;6(5):589–93
- [17] Sharma MS, Sehgal AK, Kandpal TC, Mathur SS. Geometrical–Thermal design and performance studies of a linear fresnel reflector. In: Proc. of National sol. energy conv. New Delhi: Tata McGraw Hill publishing Co. Ltd.; 1990. p. 3.032– 3.035.
- [18] Choudhury C, Sehgal HK. A fresnel strip reflector–concentrator for tubular solar-energy collectors. Appl Energy 1986; 23(2):143–54.
- [19] Rai GD. Solar energy utilization. New Delhi: Khanna Publishers; 1996. Moffat RJ. Describing the uncertainties in experimental results. Exp Therm Fluid Sci 1988; 1:3–17.
- [20] Panna Lal Singh , R.M. Sarviya , J.L. Bhagoria . Thermal performance of linear fresnel reflecting solar concentrator with trapezoidal cavity absorbers. Applied Energy 2010 (87); 541–550